DYNAMICS AND CONTROL

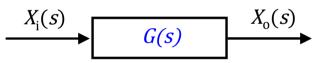
CONTROL SEMINAR 3

GENERAL INTRODUCTION – SEMINAR 3

- Block diagram manipulation
- 2nd order systems

Block Diagrams Revisited

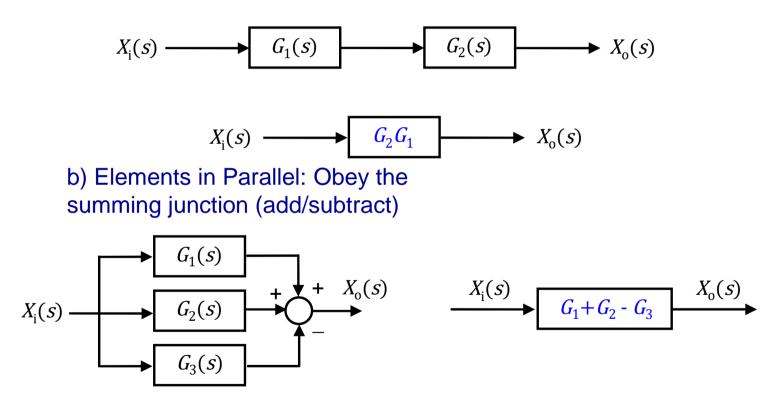
- Systems engineers represent the components of the system as a series of blocks:
- Recap:



- The transfer function G(s) transforms the input X_i into the output X₀
- $X_0(s) = G(s)X_i(s)$

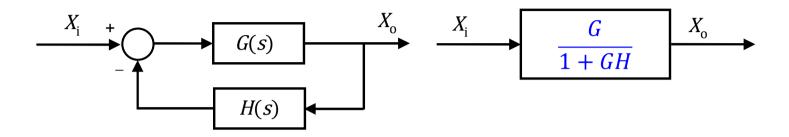
Block Diagram Manipulation: The rules

a) Elements in Series: Multiplication

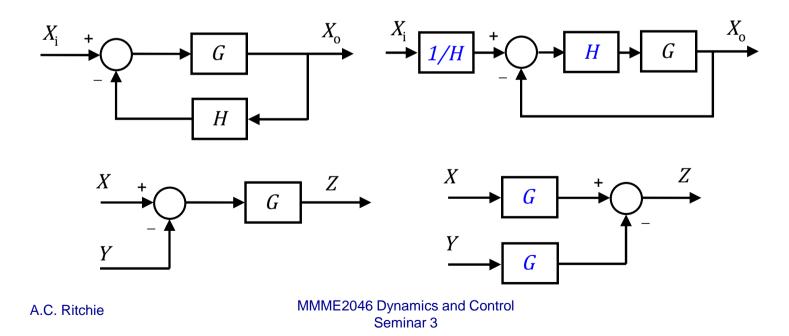


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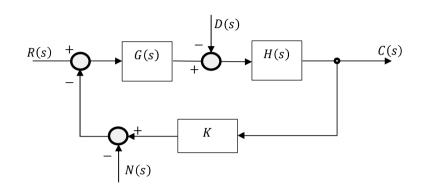
MMME2046 Dynamics and Control Seminar 3 c) Feedback (Closed loop) Transfer Function



d) Changing block positions:

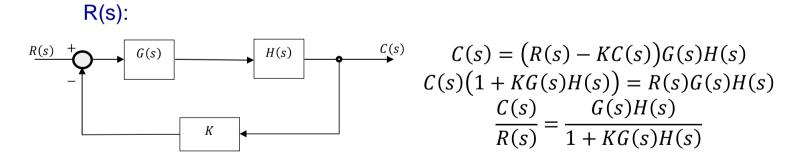


e) Dealing with disturbances



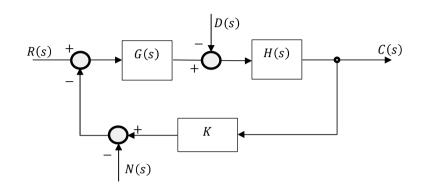
- R(s) is the input
- C(s) is the output
- D(s) is a disturbance
- N(s) is a (user controlled) compensation

- Procedure:
 - Each of the inputs has its own independent transfer function. For



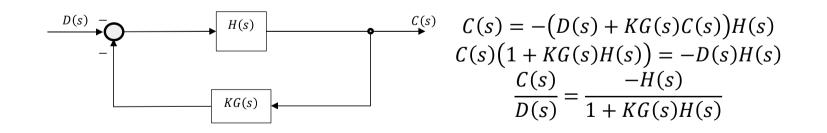
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e) Dealing with disturbances



- R(s) is the input
- C(s) is the output
- D(s) is a disturbance
- N(s) is a (user controlled) compensation

- Procedure:
 - For D(s):

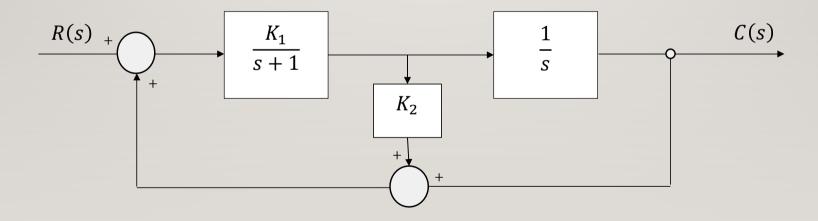


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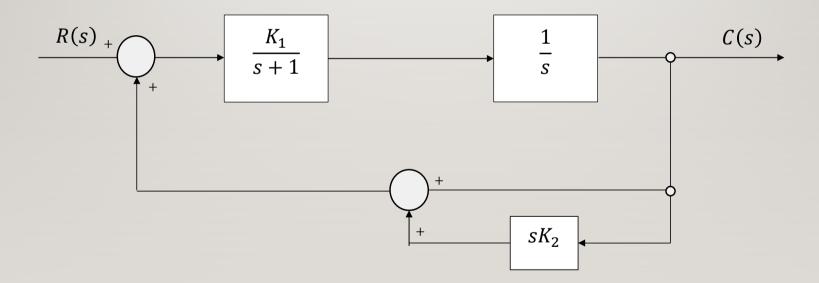
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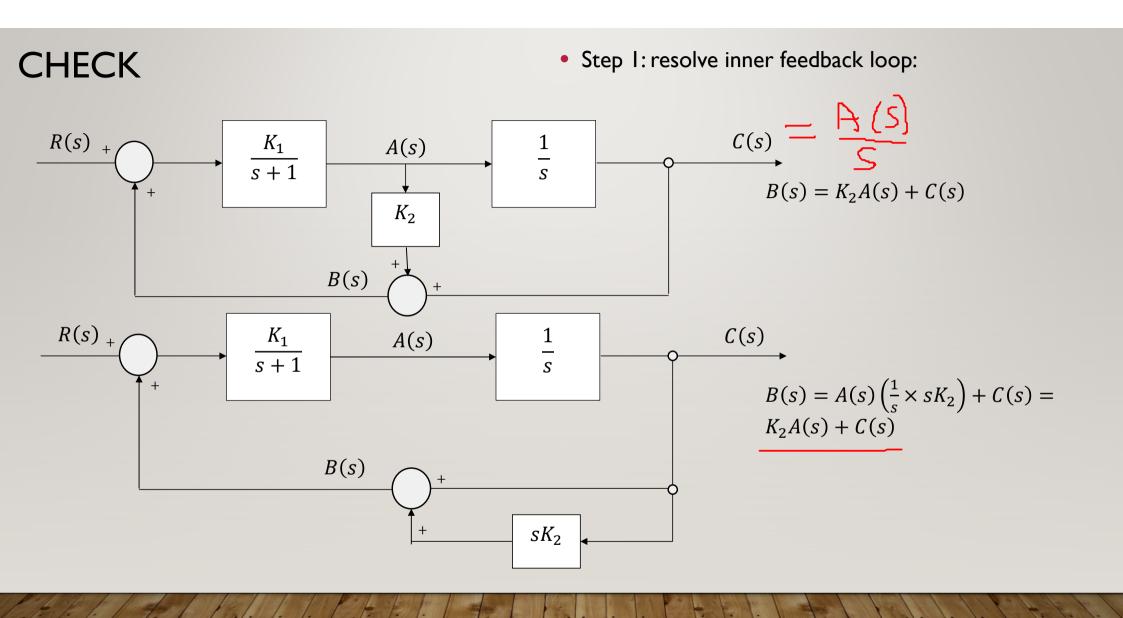
• Example 4 from Example sheet 2

Find the transfer function $\frac{C(s)}{R(s)}$ for:

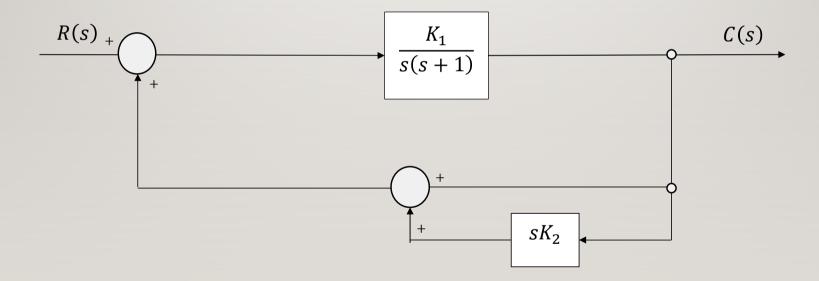


• Step I: resolve inner feedback loop:

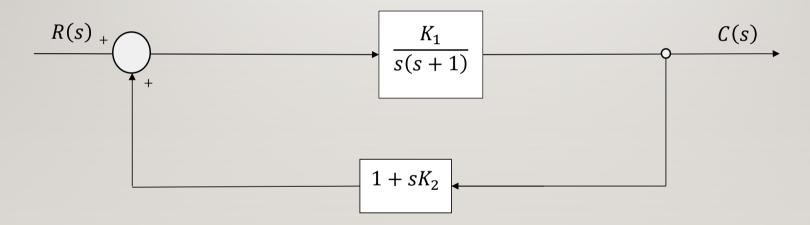


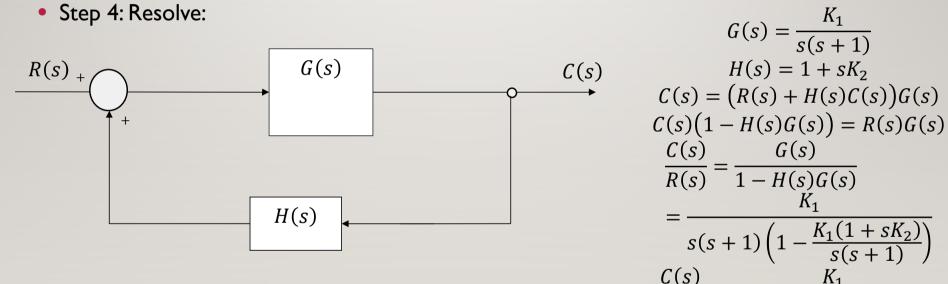


• Step 2: Combine forward transfer functions:



• Step 3: Combine feedback transfer functions:

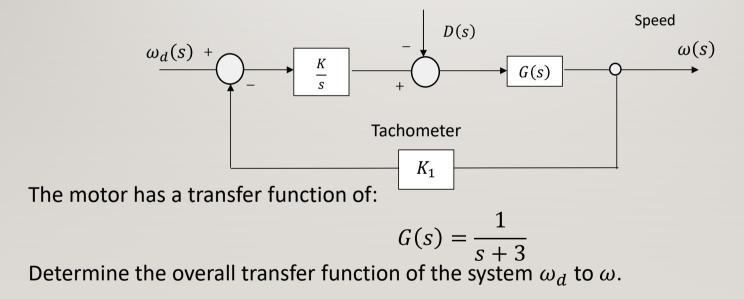




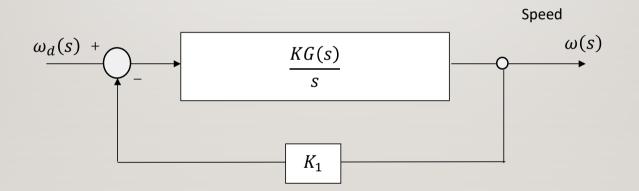
$$\frac{C(s)}{R(s)} = \frac{K_1}{s(s+1) - K_1(1+sK_2)}$$

EXAMPLE SHEET 2 QUESTION 8

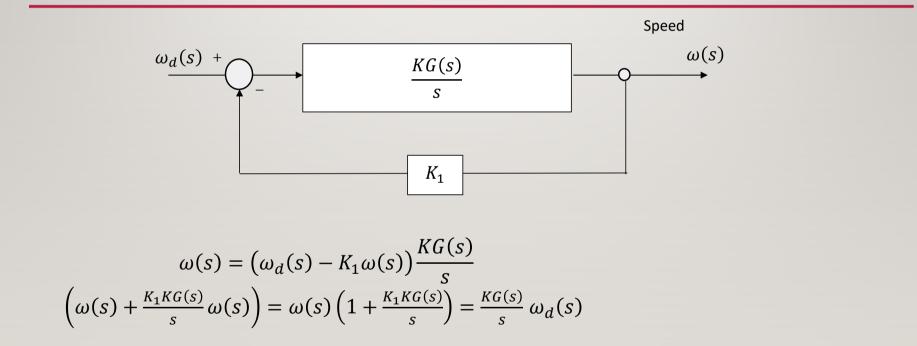
• A control system to maintain the speed of a motor is shown in figure 8.



STEP I: ELIMINATE THE DISTURBANCE AND COMBINE FORWARD TRANSFER FUNCTIONS



STEP 2: RESOLVE THE FEEDBACK LOOP



STEP 2: RESOLVE THE FEEDBACK LOOP Continued ...

$$\left(\omega(s) + \frac{K_1 KG(s)}{s} \omega(s)\right) = \omega(s) \left(1 + \frac{K_1 KG(s)}{s}\right) = \frac{KG(s)}{s} \omega_d(s)$$

We want the transfer function $\frac{\omega(s)}{\omega_d(s)}$ so:
 $\frac{\omega(s)}{\omega_d(s)} = \frac{\left(\frac{KG(s)}{s}\right)}{\left(1 + \frac{K_1 KG(s)}{s}\right)} = \frac{KG(s)}{s + K_1 KG(s)}$

2nd Order Control Systems

- 1st order systems are
 - Reliable
 - Non-oscillatory
 - Slower than 2nd order
- Hydraulics are slow and heavy
- First there was Claudia, now there is Melissa:
 - <u>https://www.youtube.com/watch?v=JWyl9RnKOIQ</u>

2nd Order Control Systems

Some examples – electro-mechanical position control



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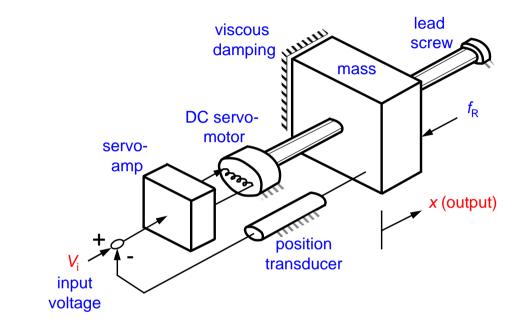
2nd Order Control Systems

- What is meant by "Second order systems"?
- You will be familiar with 2nd order differential equations and their solutions:

$$-\frac{d^2y}{dt^2} + A\frac{dy}{dt} + B = 0$$

- 2 real roots (overdamped)
- 1 root (critically damped)
- 2 complex roots (underdamped)

Example: Electro-Mechanical Position Control System



2nd order system: transfer function

$$G(s) = \frac{\omega_n^2 X_i(s)}{x^2 + 2\gamma \omega_n s + \omega_n^2}$$

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 I do not expect you to be able to solve for systems with 3rd or higher order behaviour

• Usual input is a step:
$$X_{in}(s) = \frac{A}{s}$$

• Output: $X_{out}(s) = X_{in}(s)G(s) = \frac{A}{s}\left(\frac{1}{s^2+as+b}\right)$

• Output:
$$X_{out}(s) = X_{in}(s)G(s) = \frac{A}{s}\left(\frac{1}{s^2+as+b}\right)$$

• For this system, there are 3 possible outcomes:

– Overdamped:
$$a > 2 \times \sqrt{b}$$

- Critically damped: $a = 2 \times \sqrt{b}$
- Underdamped: $a < 2 \times \sqrt{b}$

• Example:
$$X_{in}(s) = \frac{1}{s}$$
 $G(s) = \left(\frac{1}{s^2 + 5s + 4}\right)$

• Output:
$$X_{out}(s) = X_{in}(s)G(s) = \frac{1}{s}\left(\frac{1}{s^2+5s+4}\right) = \frac{1}{s}\left(\frac{1}{(s+4)(s+1)}\right)$$

- Overdamped: 2 real roots.
 - Use partial fractions to give:

$$-X_{out}(s) = \frac{1}{s} \left(\frac{1}{3(s+1)} - \frac{1}{3(s+4)} \right)$$

- Inverse Laplace Transforms (no. 8) give:

$$-x_{out}(t) = \frac{1}{3}(1 - e^{-t}) - \frac{1}{12}(1 - e^{-4t})$$

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- Example: $X_{in}(s) = \frac{1}{s}$ $G(s) = \left(\frac{1}{s^2 + 4s + 4}\right)$
- Output: $X_{out}(s) = X_{in}(s)G(s) = \frac{1}{s}\left(\frac{1}{s^2+4s+4}\right) = \frac{1}{s}\left(\frac{1}{(s+2)^2}\right)$
- Critically damped: 2 duplicate real roots.
 - Use partial fractions to give:

$$-X_{out}(s) = \frac{1}{4s} - \frac{s+4}{4(s+2)^2} = \frac{1}{4s} + \frac{s}{4(s+2)^2} + \frac{1}{(s+2)^2}$$

– Inverse Laplace Transforms give:

$$-x_{out}(t) = \frac{1}{4}(1 - te^{-2t}) - \frac{1}{4}e^{-2t}(1 - 2t) = \frac{1}{4}(1 - (1 - t)e^{-2t})$$

A.C

• Example:
$$X_{in}(s) = \frac{1}{s}$$
 $G(s) = \left(\frac{1}{s^2 + s + 4}\right)$

• Output:
$$X_{out}(s) = X_{in}(s)G(s) = \frac{1}{s}\left(\frac{1}{s^2+s+4}\right) = \frac{1}{s}\left(\frac{1}{(s+4)(s+1)}\right)$$

• Underdamped: 2 complex roots.

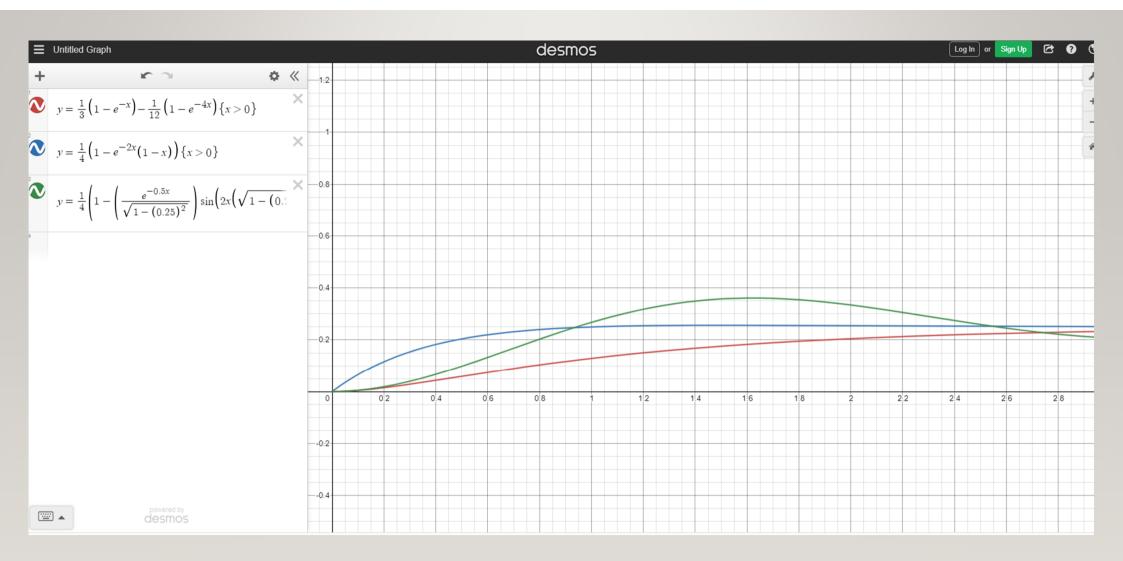
- Define
$$\omega_n = 2$$
 $2\gamma\omega_n = 1$ γ =0.25

$$-X_{out}(s) = \frac{1}{s} \left(\frac{1}{s^2 + 2\gamma\omega_n s + \omega_n^2} \right)$$

- Inverse Laplace Transforms (no. 15) give:

$$-x_{out}(t) = \frac{1}{4} \left(1 - \frac{e^{-\gamma \omega t}}{\sqrt{1 - \gamma^2}} \sin\left(\omega t \sqrt{1 - \gamma^2} + \phi\right) \right) = \frac{1}{4} \left(1 - \frac{e^{-0.5t}}{\sqrt{15/16}} \sin\left(2t \sqrt{15/16} + \phi\right) \right)$$

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THE END?

Any questions?